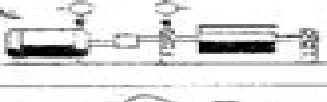
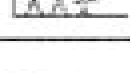


TABLE I ILLUSTRATED VIBRATION DIAGNOSTIC CHART

PROBLEM SOURCE	TYPICAL SPECTRUM	PHASE RELATIONSHIP	REMARKS
MASS UNBALANCE A. FORCE UNBALANCE			Force unbalance will be in-phase and steady. Amplitude tends to increase with the square of speed below first critical up to speed increase of 50% higher vibration. 1X, 2X, 3X peaks present and usually dominant spectrum. Can be corrected by placement of one or two correction weights in one plane at Force center of gravity (FCG). Approx. 0° phase difference should exist between CR & IS location (i.e., as well as between CR & IS vertical). Also, approx. 90° phase difference between horizontal & vertical readings usually exists at each bearing of unbalanced rotor (i.e., 0°).
B. COUPLE UNBALANCE			Couple UNB also exists at 180° out-of-phase locations on same shaft. 1X R/Rd always present and normally dominates spectrum. Amplitude varies with square of increasing speed below first critical speed. Speeds may cause high speed vibration as well as noise. Correction requires placement of corrective weights in at least 2 planes. Note that approx. 180° phase difference should exist between CR & IS horizontal, as exists between CR & IS vertical. Also, approx. a 90° difference between 180° horizontal & vertical phase readings at each bearing usually exists (i.e., 90°).
C. DYNAMIC UNBALANCE			Dynamic UNBALANCE is the dominant type of unbalance found and is a combination of both force and couple unbalance. 1X R/Rd dominates the spectrum, and may require 2 plane correction. Note: the radial phase difference between horizontal and vertical readings can vary anywhere from 0° to 180°. However, the horizontal phase difference usually closely matches the vertical phase difference, resulting in a 90° radial and vertical bearing measurements (i.e., 90°). Separately, a unbalance phenomenon, known as 90° phase difference, usually results between the horizontal and vertical readings at each bearing (i.e., 90°).
D. OVERHUNG ROTOR UNBALANCE			Overhung Rotor UNBALANCE causes high 1X R/Rd in both axial and radial directions. Axial readings tend to be in-phase unless radial phase readings might be unsteady. However, the horizontal phase difference will usually match the vertical phase difference for the horizontal inner (H/Rd). Overhung rotors have both force and couple unbalance, each of which will likely require correction. Thus, correction weights will most always have to be placed in 2 planes to counteract both force & couple unbalance.
ECCENTRIC ROTOR			Eccentricity occurs when center of rotation is offset from geometric centerline of a shaft, gear, bearing, motor assembly, etc. Largest vibration occurs at 1X R/Rd or appears in a direction of eccentricity or that two planes. Corresponding horizontal and vertical phase readings usually differ either by 0° or by 180° from which indicate centrifugal loading. Attempts to balance rotating rotors often result in reducing vibration in one radial direction, but increasing it in the other radial direction, depending on amount of eccentricity.
BENT SHAFT			Bent shaft problems cause high axial vibration with phase differences varying between 180° on the same machine components. Dominant vibration normally occurs at 1X of bent inner shaft member, but at 2X if bent outer shaft. Coupling the shafts to correct for transducer error before for such axial measurements if you reverse probe direction; i.e., use and evaluate for center bent shaft.
MISALIGNMENT A. ANGULAR MISALIGNMENT			Angular Misalignment is characterized by high axial vibration, 180° axial phase across the coupling. Usually will have high axial vibration over 2000 Hz and 2X R/Rd. However, not unusual for either 1X, 2X, or 3X to dominate. These conditions may also indicate coupling problems as well. Severe angular misalignment may cause high 1X R/Rd vibration. Unlike flexible coupling misalignment, Type B, there must be bearing noise do not typically have a steady state noise in this condition.
B. PARALLEL MISALIGNMENT			Parallel Misalignment has similar vibration symptoms to angular, but shows high axial vibration which approximates 180° out-of-phase axial coupling. 2X, often larger than 1X, due to high radial to 1X is often doubled by coupling type and construction. Symptom when Angular or Parallel Misalignment conditions exist, the dominant amplitude peaks at much higher harmonics (2X-50X), derived from forces of higher frequency than typical similar in appearance to mechanical looseness. Coupling type and material will often greatly influence the axial spectrum when misalignment is severe. Does not typically have steady state noise.
C. MISALIGNED BEARING COCKED ON SHAFT			Proper bearing fit generates substantial axial vibration. 1X R/Rd causes bearing motion with approximately 180° phase shift top to bottom and noise to side as measured in disk direction on same bearing housing. Attempts to align bearing to improve fit noise will not always reduce. Bearing usually must be removed and correctly refitted.
RESONANCE			Resonance occurs when a forcing frequency coincides with a System Natural Frequency, and can cause dramatic amplitude amplification, which might result in structural failure. This may be a natural or forced resonance of the rotor, but can also originate from support frame. Resonance peaks occur even after the rotor is at or near resonance, it can be caused by a change in the rotor or support structure. It is important to note that resonance, usually 180° when passed. Often requires changing natural frequency to a higher or lower frequency. Refused the vibration do not generally change with a change in system which may become their characteristic (i.e., large peak) bearing motion or side with amplitude significant amplification.

Vibration Analysis Guide Line

Axel Sperber

A red circular graphic with a gradient, appearing as a semi-circle or a partial circle, located to the right of the author's name.

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1. Understanding the eBook Vibration Analysis Guide Line
 - The Rise of Digital Reading Vibration Analysis Guide Line
 - Advantages of eBooks Over Traditional Books
2. Identifying Vibration Analysis Guide Line
 - Exploring Different Genres
 - Considering Fiction vs. Non-Fiction
 - Determining Your Reading Goals
3. Choosing the Right eBook Platform
 - Popular eBook Platforms
 - Features to Look for in an Vibration Analysis Guide Line
 - User-Friendly Interface
4. Exploring eBook Recommendations from Vibration Analysis Guide Line
 - Personalized Recommendations
 - Vibration Analysis Guide Line User Reviews and Ratings
 - Vibration Analysis Guide Line and Bestseller Lists
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 - Vibration Analysis Guide Line Public Domain eBooks
 - Vibration Analysis Guide Line eBook Subscription Services
 - Vibration Analysis Guide Line Budget-Friendly Options
6. Navigating Vibration Analysis Guide Line eBook Formats
 - ePub, PDF, MOBI, and More
 - Vibration Analysis Guide Line Compatibility with Devices
 - Vibration Analysis Guide Line Enhanced eBook Features
7. Enhancing Your Reading Experience
 - Adjustable Fonts and Text Sizes of Vibration Analysis Guide Line
 - Highlighting and Note-Taking Vibration Analysis Guide Line
 - Interactive Elements Vibration Analysis Guide Line
8. Staying Engaged with Vibration Analysis Guide Line

- Joining Online Reading Communities
 - Participating in Virtual Book Clubs
 - Following Authors and Publishers Vibration Analysis Guide Line
9. Balancing eBooks and Physical Books Vibration Analysis Guide Line
 - Benefits of a Digital Library
 - Creating a Diverse Reading Collection Vibration Analysis Guide Line
 10. Overcoming Reading Challenges
 - Dealing with Digital Eye Strain
 - Minimizing Distractions
 - Managing Screen Time
 11. Cultivating a Reading Routine Vibration Analysis Guide Line
 - Setting Reading Goals Vibration Analysis Guide Line
 - Carving Out Dedicated Reading Time
 12. Sourcing Reliable Information of Vibration Analysis Guide Line
 - Fact-Checking eBook Content of Vibration Analysis Guide Line
 - Distinguishing Credible Sources
 13. Promoting Lifelong Learning
 - Utilizing eBooks for Skill Development
 - Exploring Educational eBooks
 14. Embracing eBook Trends
 - Integration of Multimedia Elements
 - Interactive and Gamified eBooks

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